

HO6 – Frame Corner

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Basic Documentation – Overview

In addition to the individual program manuals, you will find basic explanations on the operation of the programs on our homepage <u>www.frilo.com</u> in the Campus-download-section.

Tip: Go back - e.g. after a link to another chapter / document - in the PDF with the key combination "ALT" + "left arrow key".



Application options

The Frame Corner application is suitable for the design of connections in rigid corners of portal frame trusses of laminated timber that are joined with dowel pins or special dowels in circular arrangement or by wedge finger jointing (one or two joints).

The verification of the corner connection resistance requires the specification of the internal forces N, M, Q acting on the frame corner. These forces result from an examination of the entire system.

Five load cases maximum can be taken into account. The verifications are carried out at selected critical points or sections.

Available standards

- DIN EN 1995-1-1:2010/2013
- ÖNORM EN 1995-1-1:2010/2015/2019
- BS EN 1995:2012
- EN 1995 (without NA)

The former standards

- DIN 1052:1996-A1
- DIN 1052:2004/2008



Basis of calculation

Dowel pins and special dowels

The calculation is based on the description in reference /1/. The loading of the dowels and the axial force are assumed to be evenly distributed over all dowels. The force applying to the dowels because of the loading by moments is proportional to the distance of the dowel from the centre of gravity of the corner joint. If the dowels are arranged in two circles in accordance with reference /3/, the permissible dowel loading may be reduced by 15 % for all dowels. The angles are always determined anti-clockwise.



The following verifications are performed:

Dowel loading

With a single ring of dowels:

exist. F =
$$\sqrt{\left(\frac{Q}{n} - \frac{M \cdot \sin(\beta_i)}{r \cdot n}\right)^2 + \left(\frac{N}{n} + \frac{M \cdot \sin(\beta_i)}{r \cdot n}\right)^2}$$

With two rings of dowels:

exist.
$$F = \sqrt{\left(\frac{Q}{n} - \frac{M \cdot r_i}{I_p} \cdot \sin(\beta_i)\right)^2 + \left(\frac{N}{n} + \frac{M \cdot r_i}{I_p} \cdot \cos(\beta_i)\right)^2}$$

The variables used:

$$\begin{split} n &= n_1 + n_2 = \text{number of dowels} \\ r_i &= \text{radius of dowel i (for i = 1, 2)} \\ \beta_i &= \text{angle coordinate of the dowel } i \\ l_p &= n_1 \cdot r_1^2 \cdot n_2 \cdot r_2^2 \end{split}$$



The dowel loading on the vertical member cross to the grain attains its maximum in point 1a or 1b, depending on the signs of M and Q. On the horizontal member, the maximum loading cross to the grain applies in point 2a or 2b. The angle between the force and the grain is decisive for the permissible dowel loading. It is fixed to 90° in the software. The distance of the dowel pins from the end-grain face should have a length of 6 times the dowel diameter, if no corner reinforcement has been provided for (the default minimum distance is $4 \cdot d_d$). All other minimum distances are specified in the standard.

In the calculation, each individual dowel is verified with its dedicated forcefibre angle.

You should ensure in this connection the proper arrangement of the dowels in the ring.



Transverse tensile stress

You can optionally verify the transversal tensile stress in the upper corner (point 3). According to reference <u>/3/</u>, this verification is in most cases only required if the permissible dowel loading and the minimum distances are not complied with or the shear stress exceeds 0.9 MN/m². The force component of a dowel perpendicular to the grain can produce cracking of the cross section in the corner area. If required, this area must be safeguarded by screw nails or screws. If two dowel rings are defined, corner reinforcement is always recommended to increase the load-bearing capacity.



III.: Maximum transverse tensile stress as per reference /1/:

exist.
$$\sigma_{Z} = 0.6 \cdot \frac{Q_{H}}{d \cdot \frac{2}{3} \cdot e}$$

Q_H = horizontal dowel force

d = timber width

e = distance of dowel to end-grain face

A force component Q_H acting in direction of the edge is more likely to produce cracking cross to the grain than a force in the opposite direction. Therefore, the force acting towards the edge is assumed positive. A force



action in the opposite direction is assumed negative. Depending on the sign of Q_{H} , the verification is based on the following condition:

$$-1 \le \text{exist.} \frac{\sigma_Z}{\text{perm.} \sigma_Z} \le +1$$

The corresponding condition applies to the horizontal member; a positive force acts from the bottom to the top.

Computing is based on the most unfavourable assumption that horizontal loading of a dowel produced by the shear force component acts in the same direction as horizontal loading on that dowel produced by moments. The minimum increase of the distance to the edge *e* due to an inclination of the horizontal member is not taken into account.

An inclination angle, a displacement of the dowels in the ring or load portions with opposite signs may produce a situation in which the decisive dowel is not the one with the lowest distance to the edge.

Because all dowels are verified individually, their respective force components are known and a more accurate and economic verification can be performed.

Tests have revealed that frame corners fail in most cases when the shear or transverse tension limit is attained in the corner area. Therefore, Heimeshoff recommends in reference /3/ the design of a corner reinforcement for a force $N_D = n_1/12 \cdot D_M$ (D_M = dowel force caused by a moment) if two dowel rings should be installed.

Shear stress resistance verifications

The maximum shear forces in the horizontal and vertical members occur in section I-I and II-II:

Vertical member in section I-I:	Q I-I	= Q/2 - M/r/ π
Horizontal member in section II-II:	Q -	= N/2 - M/r/π

If you have defined two dowel rings, replace the second term by:

$$\frac{M}{\pi} \cdot \frac{n_1 \cdot r_1 + n_2 \cdot r_2}{n_1 \cdot r_1^2 + n_2 \cdot r_2^2}$$

Shear stress:

TauQ I-I = $3/2 \cdot Q$ I-I /A I-I TauQ II-II = $3/2 \cdot Q$ II-II/A II-II

Since the shear stress inside the dowel ring is decisive, the properties of the unweakened cross section are used.

$$\frac{\text{exist. } \tau_{\text{Q}}}{\text{perm. } \tau_{\text{Q}}} \leq 1$$

The software uses a permissible Tau Q = 1.2 MN/m^2 in the calculation deviating from the recommendation in reference /3/. In combination with DIN 1052:2008 and EN 1995, $f_{v,d}$ is used. The stresses and loading are verified for each load case.

Torsion spring stiffness

The torsion spring stiffness is determined with the help of the expression:

$$K_{m,mean} = \frac{2}{3}K_{ser} \cdot I_p$$

taking the displacement module K_{Ser} into account.



Verifications with wedge finger joints

The calculation is based on reference $\frac{2}{2}$. In wedge finger connections with one or two wedge finger joints, the stress distribution with a negative corner moment is as follows:



III.: Stress distribution

On typical frames with a negative corner moment, the resistance to the high compressive stresses applying at the inner edge of the corner must be verified.

As a positive corner moment produces transverse tensile stress and the load-bearing behaviour is largely unknown, this type of wedge finger joint should not be used if larger positive corner moments apply. Until clarification of the load bearing, the permissible longitudinal tensile stress will be reduced to 20 % of σ_D as recommended by reference /2/.

The following applies to the stress resistance verifications in the horizontal and vertical members:

exist. $\sigma_{D}(\gamma) = \omega \cdot \frac{N}{A_{n}} + \frac{\text{perm.}\sigma_{DII}}{\text{perm.}\sigma_{B}} \cdot \frac{M}{W_{n}}$

Because the signs are defined negative if pressure applies and positive if tension applies, the following conditions must be satisfied in the verifications:

With pressure: exist. $\sigma_D(\gamma) = \text{perm. } \sigma_D(\gamma) \text{ MN/m}^2$ With tension: exist. $\sigma_7(\gamma) = 0, 2 \cdot \text{perm. } \sigma_D(\gamma) \text{ MN/m}^2$

The permissible compressive stress is set to the value applicable for coniferous timber S10 by default.

 γ is the angle between the force and the grain. N, M are the internal forces, converted by the software, that are referenced to the centre of the wedge finger joint. A_n and W_n refer to the decisive net cross section properties at the frame corner perpendicular to the member axis, which are reduced to the 0.8-fold values of the gross cross sectional properties due to the weakening effect of the wedge finger joint.



Calculation of the internal design forces with two wedge finger joints:

Joint vertical member: $Q_s = Q, N_s = N, M_s = M - Q \cdot I'$ Joint horizontal member: $Q_r = N \cdot sin(\gamma) + Q \cdot cos(\gamma)$ $N_r = N \cdot cos(\gamma) - Q \cdot sin(\gamma)$ $M_r = M - N \cdot I \cdot cos(\alpha) + Q \cdot I \cdot sin(\alpha)$

with I' = 0,5 · (h +
$$\frac{a}{\tan(\gamma)}$$
) · $\tan(2 \cdot \gamma) - \tan(\gamma)$

In the design, the verification as per DIN EN 1995 is implemented as follows:

$$\frac{f_{c,0,d}}{f_{c,\gamma,d}} \cdot \left(\frac{\sigma_{c,0,d}}{k_c \cdot f_{c,0,d}} + \frac{\sigma_{m,d}}{f_{m,d}}\right) \le 1 \text{ with } f_{c,\alpha,d} = \frac{f_{c,0,d}}{\sqrt{\left(\frac{f_{c,0,d}}{2 \cdot f_{c,90,d}} \sin^2 \alpha\right)^2 + \left(\frac{f_{c,0,d}}{2 \cdot f_{v,d}} \sin \alpha \cdot \cos \alpha\right)^2 + \cos^4 \alpha}}$$

For the strength levels $f_{m,k}$, always the next lower strength class should be used.

The strength levels $f_{m,k} / f_{c,k}$ are reduced by 15% for softwood or glulam of strength level GL 24 or higher. The strengths fm, k / fc, k are reduced by 15% for needle or glulam timber of strength class GL 24 or higher.

You should estimate the buckling coefficients $k_{\rm c}$ for the vertical and horizontal members in accordance with the given conditions and set them manually.

As recommended by reference $\frac{2}{2}$, a permissible stress of 20 % is considered in the calculation if the corner moments are positive.



Definition of the structural system

Select first the desired connection:

- Dowel ring
- Wedge finger joint

The corresponding definition dialog is displayed.

For the geometry of the system, it is assumed that the grains in the horizontal and vertical members run in parallel to the outer member edges. Slanted interior edges do not affect the calculation in any way. The corner area to be examined is described by a characteristic cross section height *h* for the vertical and horizontal members (perpendicular to the member axis). As the cross section dimensions in the frame corner are determined by the connection exclusively, we recommend estimating the actual cross section axes and the location of the centre of rotation as exact as possible.

Material selection

You can select the sorting class directly.

To enter the usage class and the material coefficients (specific weight Gamma), click on the button.

Note: Softwood and hardwood according to EN 338: 2016 implemented. Glulam according to EN 14080: 2013 implemented for Germany and Italy. The "old" glulam timbers are marked with an asterisk (e.g., GL24c *).

timber acc. to DIN EN 1995-1-1/N	A:2013-08
timber sort	strength-class
Glulam	GL30c (EN 14080:2013) GL30h (EN 14080:2013) GL32c (EN 14080:2013) GL32h (EN 14080:2013) GL24c* (DIN 1052:2008) GL24h* (DIN 1052:2008) GL28c* (DIN 1052:2008)
service class specific weight	gamma = <u>5,00</u> kN/m3

material

GL24h (EN 14080:2013 🔻



Dowel ring

System definition - dowel rings

Dowel pins should be preferred to special dowels because they are easier to install and can bear higher loading. Dowelled frame corners must be protected against shrinkage cracking if they are installed outdoors or in locations with high climatic variations. This is of particular importance in combination with large dowel rings that require great cross section dimensions.

Alpha	inclination of the horizontal member, angle between the	system		
	horizontal axis and the member edge	Alpha=	20,0	degree
Vertical member hs	height of vertical member	strut hs=	80,0	cm
Horizontal member hr	height of horizontal member	bar hr=	80,0	cm
	neight of honzontal member	strut bs=	12,0	cm (2x)
Vertical member bs	width of vertical member	bar br=	22,0	cm
Horizontal member	width of horizontal member	eu=	8,5	cm
eu	existing projection			



Ill.: Circular dowel arrangement (dowel rings) on a frame corner



You must define distances for the dowel arrangement.

Projection:	eu	≥ 0 cm
Distance to the edge:	e _R =	$\frac{h_s - 2 \cdot r_1}{2}$
Distance to the edge:	e _R	special dowels $\geq b/2$
		pin dowels $\ge 3 \cdot d_d$

The distance $(e_U + e_R)$ to the grain-end face of the dowels close to the edge must be complied with: dowel pins $\ge 4 \cdot d_d$ (slanted connection with an angle of 60° approx.) and special dowels $\ge e_d$.

If no corner reinforcement is provided for, pin dowels should have a minimum spacing of $6 \cdot d_d$ in accordance with reference /3/. For the distance e of the dowel at point 3 to end-grain face, which is decisive for the transverse tension resistance verification, the following applies:

 $e = e_{U} + e_{R} + r_{1} \cdot (1 - \sin\beta + \cos\beta \cdot \tan\alpha)$

 β = angle coordinate of dowel 3 = 135° $\,$ + $\alpha/2$

Fasteners - dowel selection

Check either the Dowel pin or Special dowel option and click to the "Dowel" button to display the dialog for the selection of the fastener type.

Circular arrangement

The rotation angle for the circular arrangement should be defined in the anti-clockwise direction. The specifications are immediately shown on the graphic screen.

Design

	4	design	
1 / 2 rings	select two rings only if a solution with a single ring is not appropriate for structural reasons or because of the geometry. The permissible dowel load is reduced when defining two rings.	1 circle 2 circles out.circle radius= 34,5 max = 17	auto. design Eta = 1,232 cm radius= 20,5 cm pcs max = 10 pcs
Outer ring	the outer ring is defined by its radius and the number of dowels. If the cross section dimensions are given, you can increase the radius	sel = 12 s by selecting a pro	pcs sel = 0 pcs ojection " <u>eu</u> ".
Max.	indicates the maximum number of fasteners that	you can arrange g	eometrically in the ring.
Req.	indicates the structurally required number of faste	eners to be arrange	ed in the ring.
Sel.	specification of the (selected) number of dowels i	in the ring.	
Inner ring	the definition options for the inner ring are enable	d when you activat	te the two rings option.
Eta	displays the maximum utilization ratio of the dow	el and the cross se	ection.



Wedge finger joint

System definition for wedge finger joints

Double-clicking on the corresponding item in the main menu displays the dialog for the definition of the wedge finger joint. Select the number of joints (1 / 2 joints)

Alpha	inclination of the horizontal member, angle between the horizontal axis and the member edge
h	cross section height of the vertical and horizontal members in the corner area
b	width of the horizontal and vertical members
а	length of the intermediate section (with two joints)
kc,S	buckling coefficient of vertical member
kc,R	buckling coefficient of horizontal member







If an intermediate section was defined, the grain angle is halved. This increases the permissible stresses.

The forces generated through the change of direction in the component are assumed to produce only lateral pressure (this is true with negative corner moments). In the structurally ineffective area of the corner, no bracing should be connected.

The required buckling coefficients for the horizontal and vertical members shall be estimated if no accurate calculation is carried out. The most unfavourable value for buckling in/out of the frame plane shall be taken into account in each case.



Design

The existing stresses in the lower fibre and the utilisation ratio are displayed for the horizontal and vertical members.

Loads / internal design forces

Specify the internal design forces

Nd axial force

Md moment

with the associated class of the load-action period "KLED" in the ultimate limit state.

Des. sit. design situation: you can optionally select the accidental design situation. In this case, the material coefficient γ_M = 1.0 is used in the calculation.

$$R_{d} = R_{k} \cdot \frac{K_{mod}}{\gamma_{M}}$$

internal for	ces	
Vd=	-24,00	kN
Nd=	-40,00	kN
Md=	-82,00	kNm
LDC	medium-te	-
des.sit.	permanent	-
	permanent/	'transient
fastenings	accidental	6
🔘 bar dow	el	
Odded to the od	of special kin	d
	dowel]
circle-posit	ion	
rotation	= (),0 degree

Design load cases

Activating the item "Design LC" in the main menu accesses the load case definition dialog. You can define five load cases maximum.



Design options

Activating the "Options" menu item accesses the design options dialog.

For special dowels type C, you can optionally take the load-bearing capacity of the bolt into account.

The verification of the resistance to transverse tensile stress is optionally available for individual dowels.



Output

The user can launch the output of system data, results and graphical representations on the screen or the printer via the Output menu item in the main menu.

Extensive output	the different scopes of the normal and the extensive output are described in the following chapter (legend).
Word	If installed on your computer, the text editor MS Word is launched and the output data are transferred. You can edit the data in Word as required.
Screen	displays the values in a text window on the screen
Printer	starts the output on the printer
Page view	(file menu) displays a <u>Print preview</u> .

Legend of output codes

Output of the dowel loading

In the standard output, only the decisive dowel is put out for each load case, whereas the extensive output includes all dowels for all load cases.

dov lc	D.No	ading out Ftot,d	tside βtot	0s [Grad]	0(R [Grad]	Rd,c	Rd, b	η	
1	1	11.613	85.1	4.9	65.1	9.426	0.000	1.232	
LC		load case number							
D.no.		dowe	l number						
$F_{tot,d}$		total r	esultant of	the dowe	l loading				
β_{tot}		assoc	iated direc	tion of loa	iding				
$\alpha_{\rm S}$		force-	fibre angle	of the res	ulting dow	el loading	in the ver	rtical membe	er
α_{R}		force-	fibre angle	of the res	ulting dow	el loading	in the ho	rizontal men	nber
$R_{d,C}$		resist	ing load of	the dowel					
$R_{d,b}$		resist	ing load of	the bolt, if	available	(depends	on the do	wel type)	
η		utiliza	ition ratio						

Output of the distances to the edges

In the standard output, only the existing minimum and required maximum distances are put out.

In the extensive output, the existing distances and the minimum values among them are put out for each dowel in the first row.

	edge di	istance to strut	loaded	edge bar					
	no	a90,5 [cm]	ao,s [cm]	а90,R [cm]	ао, к [cm]	adub [cm]	es [cm]	er [cm]	
	italic sho	wn values	are the red	uired min.	values.				
	1	5.5	51.1	51.8		18.1	62.9	88.3	
	2	4.0 69.9	9.8 32.7	5.1 34.0	83.4	10.0 18.1	44.0	83.4	
Ν	0.	dowel number							
а	90,S	dista	ance to th	ne loaded	edge perpe	endicular t	o the grair	n in the verti	cal m
а	0,S	distance to the loaded edge parallel to the grain in the vertical member							
а	90,R	dista	ance to th	ne loaded	edge perpe	endicular t	o the grair	n in the horiz	zonta
а	0,R	distance to the loaded edge parallel to the grain in the horizontal member							
а	Dub	spacing of the dowels							
e	S	dista	ance to th	ne end-gra	in face in t	he vertica	l member		
е	R	dista	ance to th	ne end-gra	in face in t	he horizor	ntal memb	er	



Output of the transverse and longitudinal axial forces

In the extensive output, the transverse and longitudinal axial forces in the dowels are put out in addition.

transversal and longitudal forces in dowels

	lc d.i	no	dx	dy	Ftot	Beta	FHtot	FVtot	FHM	FVM
	1	1	0.34	0.00	11.61	85.1	1.00	11.57	0.00	9.90
L	C		load	case n	umber					
C).no.		dowe	el num	ber					
С	lх		dista	ince to	the ring of	centre i	n the ho	rizontal o	direction	í
С	ly		dista	ince to	the ring of	centre i	n the ve	rtical dire	ection	
F	tot		total	resulta	ant of the	dowel	loading			
E	Beta		asso	ciated	angle of	the res	ultant			
F	Htot		horiz	ontal f	orce com	nponen	t of the r	resultant		
F	Vtot		verti	cal for	ce compo	nent of	f the res	ultant		
F	ΉM		horiz	contal f	orce com	nponen	t resultir	ng from tl	he mom	ent portion
F	VM		verti	cal for	ce compo	nent re	sulting	from the	moment	t portion

Since the axial force and shear force portions are distributed constantly, they are not represented in the table . You can quickly calculate this value from the values shown in the table by subtracting the moments portion from the resultant: e.g. FH(Q)=FHtot-FHM.

Output of the transverse tensile stress

In the standard output, only the decisive dowels are put out; in the extensive version, all dowels are put out.

Que	erzu	ıgspar	nnung								
LF	D.	Nr	es [cm]	e _R [cm]	σ _{t90ds} [kN/cm²] [l	ot90dR kN/cm²]	ηs	η _R			
1 1		2 6	34.5 69.1	75.9 27.6	0.0002 0.0030	0.0060 0.0002	0.007 0.098	0.196 0.005]	
LC			load	d case	number						
D.n	0.		dov	el nur	nber						
es			dist	ance c	of the exa	imined d	owel to t	the end-	grain edg	e of the vertical member	
e _R			dist	ance c	of the exa	imined d	owel to t	the end-	grain edg	e of the horizontal member	
σ_{t90dS} transverse tensile stress for 60 % of the dowel lo					vel loadin	g perpendicular to the grain distributed					
			ove	r 2/3 c	of the area	a with es	in the ve	ertical m	ember		
σ_{t90dR}		R	transverse tensile stress for 60 % of the dowel loading perpendicular to the grain distributed								
			ove	r 2/3 c	of the area	a with <i>es</i>	in the he	orizonta	member		
η_{S}			utili	zation	ratio in t	he vertic	al memb	ber			
η_{R}			utilization ratio in the horizontal member								

Output of the forces to be borne by the fasteners

lc	Faxd,S [kN]	Faxd,R [kN]	
1	9.90	19.81	
LC	load	case numb	er
F _{axd,S}	tensil	e force to I	be borne by the fasteners in the vertical member
$F_{axd,R}$	tensil	e force to I	be borne by the fasteners in the horizontal member



Reference literature

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